



초공동 유동해석 기술

Numerical Analysis of Supercavitating Flow

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2019.04.23

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1. Introduction
2. Governing equations & Numerical method
3. Results and discussion
4. Conclusions

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Cavitation

- Widely observed in various propulsion systems and high-speed underwater objects, such as marine propellers, impellers of turbomachinery, hydrofoils, nozzles, injectors and torpedoes.

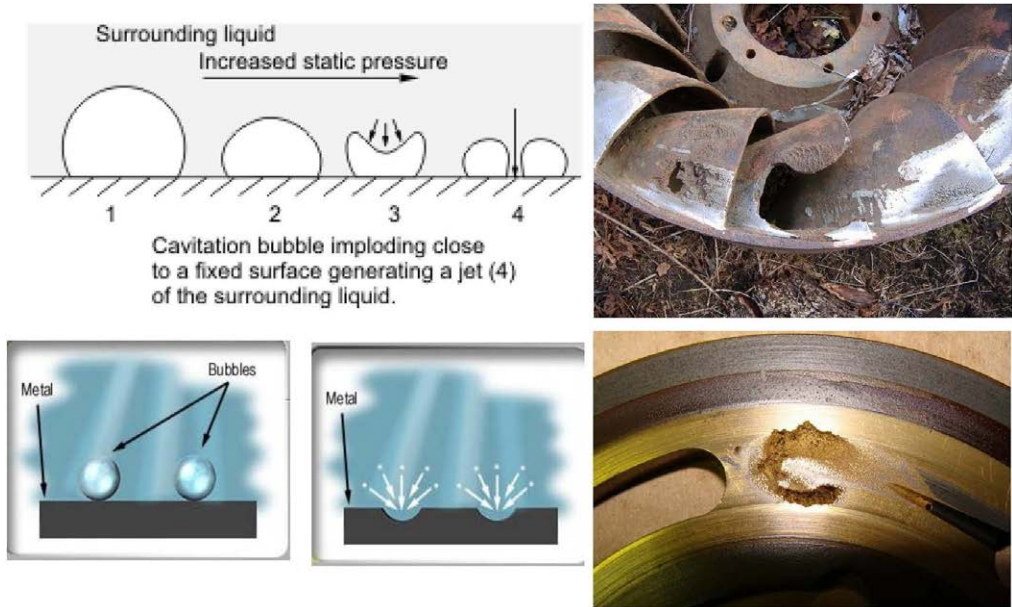


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Erosion by Cavitation Bubble



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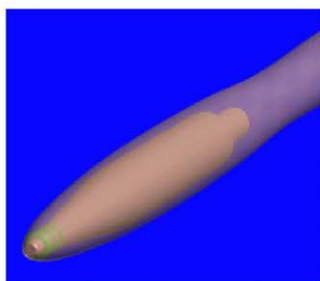
Drag Reduction

- Torpedo Speeds
 - Mk-46 Torpedo : 45 knots (80km/h)
 - Shkval I : > 250 knots (450km/h)
 - Shkval II : > 350 knots (rumored 720 knots)
 - Barracuda : > 360 knots (rumored 800 knots)
1,440km/h



MK-46

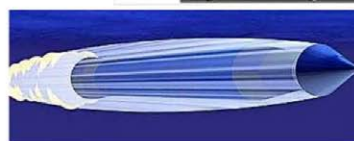
초월공동어뢰



ONR/PSU ARL



Shkval(Squall)

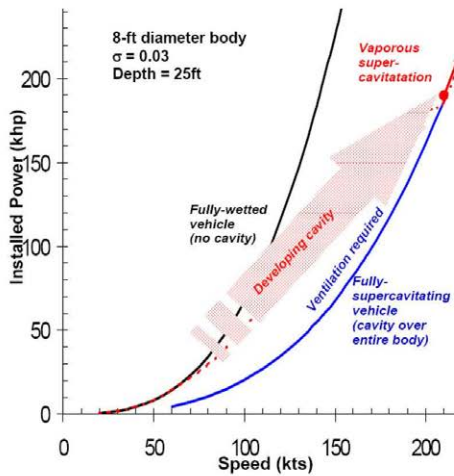


Barracuda
2005 American Controls Conference
Portland, OR June 10, 2005

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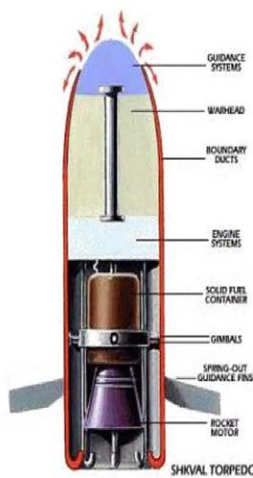
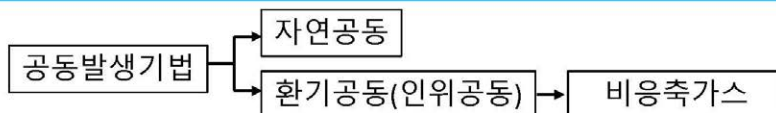
Drag Reduction



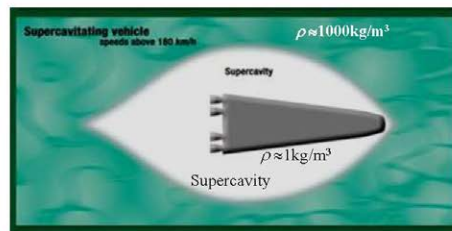
Drag Reduction Approaches	Speed +	Mechanism
Shaping	10%	Reduces form and wave drag
Polymers	20%	Reduces boundary layer frictional drag
Microbubbles	12%	Reduces boundary layer frictional drag
Coatings	4%	Modifies frictional drag of boundary layer
Supercavitation	65%	Eliminates most frictional drag



Drag Reduction



< Shkval-I Torpedo >



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Drag Reduction

Underwater Express / SST (Super-fast Submerged Transport)

Speed ~100 knots

Size : 8 ft dia., 60 tones

- Comparable in size to current special purpose craft such as the MK V Special Operations Craft and the Advanced SEAL Delivery Vehicle

- Mark V: 82 feet long aluminum monohull surface craft, 40 knots

Demonstrate stable and controllable high-speed underwater transport through supercavitation for future littoral missions

Period : Nov. 2007 ~ 2010

Monitored by DARPA / ATO

Northrop Grumman Team selected

(\$45.8 million for 3 phases, ~4 years)

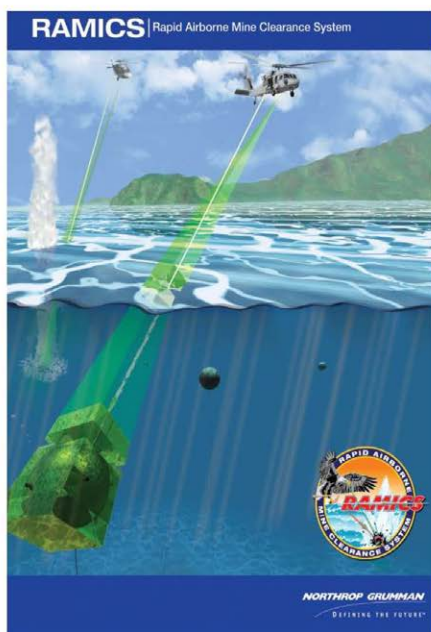


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Drag Reduction

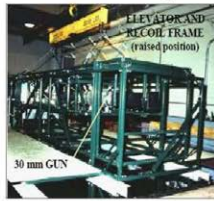


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Supercavitating bullet

30 mm Underwater Firing Fixture



Typical Launch




Safety Containment



Undersea Gun

NUWC UNDERWATER SUPERCAVITATING HIGH-SPEED BODIES TEST FACILITY

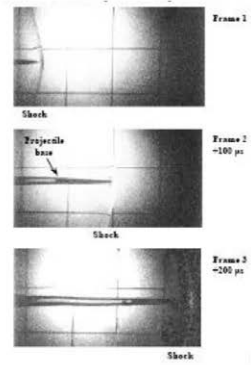


VERTICAL VIEW

$\sigma = O(10^{-3})$, $V = 1220 \text{ m/s}$
($a_{\text{water}} = 1480 \text{ m/s}$)

DISCOVERY


TUNGSTEN PROJECTILE, 1100 m/s

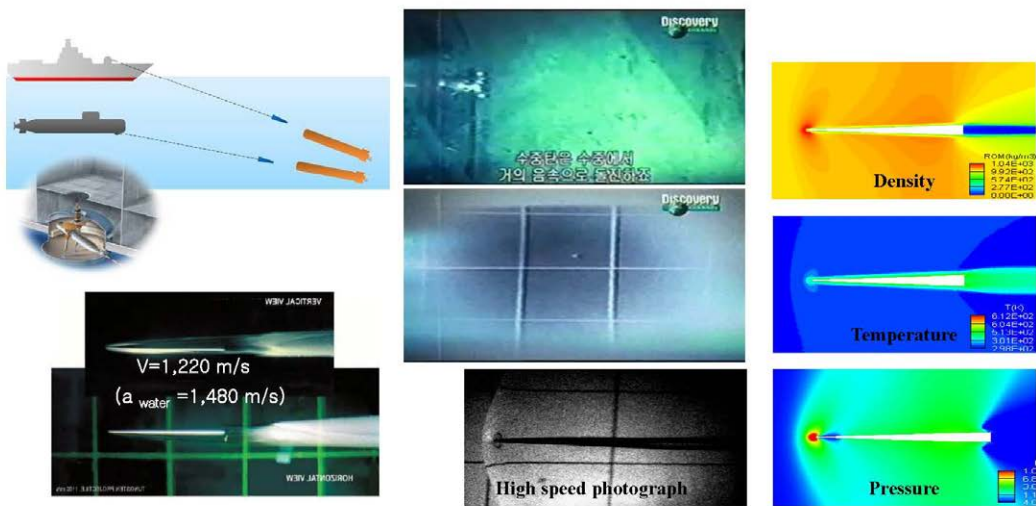


Supercavitating bullet



Supercavitating bullet

 High-Speed Underwater Bullet : $U_{\infty}=1,540$ km/h ($M_{\infty}=1.03$); $P_{\infty}=152.0$ kPa (Depth : 4m); $T_{\infty}=27^{\circ}\text{C}$
 - AHSUM (Adaptable High-Speed Underwater Munitions)



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지배 방정식 (Governing Equation)

□ Continuity Equation

$$\frac{\partial}{\partial t}(Y_l \rho_m) + \frac{\partial}{\partial x}(Y_l \rho_m u) + \frac{\partial}{\partial y}(Y_l \rho_m v) + \frac{\partial}{\partial z}(Y_l \rho_m w) = [(\dot{m}_p^+ + \dot{m}_p^-) + (\dot{m}_T^+ + \dot{m}_T^-)]$$

$$\frac{\partial}{\partial t}(Y_v \rho_m) + \frac{\partial}{\partial x}(Y_v \rho_m u) + \frac{\partial}{\partial y}(Y_v \rho_m v) + \frac{\partial}{\partial z}(Y_v \rho_m w) = -[(\dot{m}_p^+ + \dot{m}_p^-) + (\dot{m}_T^+ + \dot{m}_T^-)]$$

$$\frac{\partial}{\partial t}(Y_s \rho_m) + \frac{\partial}{\partial x}(Y_s \rho_m u) + \frac{\partial}{\partial y}(Y_s \rho_m v) + \frac{\partial}{\partial z}(Y_s \rho_m w) = 0$$

□ Momentum Equation

$$\frac{\partial}{\partial t}(\rho_m u) + \frac{\partial}{\partial x}(\rho_m u^2) + \frac{\partial}{\partial y}(\rho_m uv) + \frac{\partial}{\partial z}(\rho_m uw) = -\bar{p}_p \frac{\partial p}{\partial x} + \frac{\partial \tau_{xx}}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} + \rho_m g_x$$

$$\frac{\partial}{\partial t}(\rho_m v) + \frac{\partial}{\partial x}(\rho_m vu) + \frac{\partial}{\partial y}(\rho_m v^2) + \frac{\partial}{\partial z}(\rho_m vw) = -\bar{p}_p \frac{\partial p}{\partial y} + \frac{\partial \tau_{yx}}{\partial x} + \frac{\partial \tau_{yy}}{\partial y} + \frac{\partial \tau_{yz}}{\partial z} + \rho_m g_y$$

$$\frac{\partial}{\partial t}(\rho_m w) + \frac{\partial}{\partial x}(\rho_m wu) + \frac{\partial}{\partial y}(\rho_m wv) + \frac{\partial}{\partial z}(\rho_m w^2) = -\bar{p}_p \frac{\partial p}{\partial z} + \frac{\partial \tau_{zx}}{\partial x} + \frac{\partial \tau_{zy}}{\partial y} + \frac{\partial \tau_{zz}}{\partial z} + \rho_m g_z$$

□ Energy Equation

$$\frac{\partial}{\partial t}(\rho_m h_l - \bar{p}_p p) + \frac{\partial}{\partial x}(\rho_m h_l u) + \frac{\partial}{\partial y}(\rho_m h_l v) + \frac{\partial}{\partial z}(\rho_m h_l w) = \frac{\partial(u\tau_{xx} + v\tau_{xy} + w\tau_{xz} - q_x)}{\partial x} + \frac{\partial(u\tau_{yx} + v\tau_{yy} + w\tau_{yz} - q_y)}{\partial y} + \frac{\partial(u\tau_{zx} + v\tau_{zy} + w\tau_{zz} - q_z)}{\partial z} - h(\dot{m}_T^+ + \dot{m}_T^-)$$

Phase change due to pressure : **cavitation**

Phase change due to temperature

$$\dot{m}_T^+ = C_{tp} \rho_v \alpha_v \max\left(\frac{T_v - T}{T_v}, 0\right)$$

$$\dot{m}_T^- = C_{tl} \rho_l \alpha_l \min\left(\frac{T_v - T}{T_v}, 0\right)$$



지배 방정식 (Governing Equation)

$$\Gamma_e \frac{\partial \hat{Q}}{\partial t} + \Gamma \frac{\partial \hat{Q}}{\partial \tau} + \frac{\partial(\hat{E} - \hat{E}_v)}{\partial \xi} + \frac{\partial(\hat{F} - \hat{F}_v)}{\partial \eta} + \frac{\partial(\hat{G} - \hat{G}_v)}{\partial \zeta} = \hat{S}$$

Isothermal	Fully Compressible
$\Gamma_e = \begin{pmatrix} 0 & 0 & 0 & 0 & 0 \\ 0 & \rho_m & 0 & 0 & u \Delta \rho_1 \\ 0 & 0 & \rho_m & 0 & v \Delta \rho_1 \\ 0 & 0 & 0 & \rho_m & w \Delta \rho_1 \\ 0 & 0 & 0 & 0 & 1 \end{pmatrix}$	$\Gamma_e = \begin{pmatrix} Y_l \partial_p \rho_m & 0 & 0 & 0 & Y_l \partial_T \rho_m & -\rho_m + Y_l \partial_{Yv} \rho_m \\ u \partial_p \rho_m & \rho_m & 0 & 0 & u \partial_T \rho_m & u \partial_{Yv} \rho_m \\ v \partial_p \rho_m & 0 & \rho_m & 0 & v \partial_T \rho_m & v \partial_{Yv} \rho_m \\ w \partial_p \rho_m & 0 & 0 & \rho_m & w \partial_T \rho_m & w \partial_{Yv} \rho_m \\ ht \partial_p \rho_m + \rho_m \partial_p h - 1 & \rho_m u & \rho_m v & \rho_m w & ht \partial_T \rho_m + \rho_m \partial_T h & ht \partial_{Yv} \rho_m + \rho_m \partial_{Yv} h \\ Y_v \partial_p \rho_m & 0 & 0 & 0 & Y_v \partial_T \rho_m & \rho_m + Y_v \partial_{Yv} \rho_m \end{pmatrix}$
$\Gamma = \begin{pmatrix} 1/\beta^2 & 0 & 0 & 0 & 0 \\ 0 & \rho_m & 0 & 0 & u \Delta \rho_1 \\ 0 & 0 & \rho_m & 0 & v \Delta \rho_1 \\ 0 & 0 & 0 & \rho_m & w \Delta \rho_1 \\ \alpha_l/\beta^2 & 0 & 0 & 0 & 1 \end{pmatrix}$	$\Gamma = \begin{pmatrix} Y_l \partial_p \rho_m & 0 & 0 & 0 & Y_l \partial_T \rho_m & -\rho_m + Y_l \partial_{Yv} \rho_m \\ u \partial_p \rho_m & \rho_m & 0 & 0 & u \partial_T \rho_m & u \partial_{Yv} \rho_m \\ v \partial_p \rho_m & 0 & \rho_m & 0 & v \partial_T \rho_m & v \partial_{Yv} \rho_m \\ w \partial_p \rho_m & 0 & 0 & \rho_m & w \partial_T \rho_m & w \partial_{Yv} \rho_m \\ ht \partial_p \rho_m + \rho_m \partial_p h - 1 & \rho_m u & \rho_m v & \rho_m w & ht \partial_T \rho_m + \rho_m \partial_T h & ht \partial_{Yv} \rho_m + \rho_m \partial_{Yv} h \\ Y_v \partial_p \rho_m & 0 & 0 & 0 & Y_v \partial_T \rho_m & \rho_m + Y_v \partial_{Yv} \rho_m \end{pmatrix}$



지배 방정식 (Governing Equation)

$$\Delta \hat{Q}^{n+1,k} = \hat{Q}^{n+1,k+1} - \hat{Q}^{n+1,k} \hat{R}^{n+1,k} = \frac{3\hat{Q}^{n+1,k} - 4\hat{Q}^n + \hat{Q}^{n-1}}{2\Delta t} + \left[\left(\frac{\partial \hat{E}}{\partial \xi} + \frac{\partial \hat{F}}{\partial \eta} \right) - \left(\frac{\partial \hat{E}_v}{\partial \xi} + \frac{\partial \hat{F}_v}{\partial \eta} \right) + \hat{S} \right]^{n+1,k}$$

$$\left[I + \Delta t \Gamma^{-1} \left(\frac{\partial A}{\partial \xi} + \frac{\partial B}{\partial \eta} + 1.5 \frac{\Gamma_e}{\Delta t} - S \right) \right] \Delta \hat{Q}^{n+1,k} = -\Delta t \Gamma^{-1} \hat{R}^{n+1,k}$$

- Second-order accurate backward difference for the physical time derivative
- First order upwind for the convective flux terms (implicit part)
- First-order backward difference for the pseudo time derivative
- Upwind Non-MUSCL Total Variation Diminishing (TVD) approach for the convective flux terms (Residual part)
- A second-order central for the viscous flux terms (Residual part)

Cavitation & Boiling Models

Cavitation Models

Option #1 : Merkle et al. 1998

Option #2 : Lindau et al. 2002

$$\dot{m}^- = \frac{C_{dest} \rho_l \alpha_l \min[0, p - p_v]}{(\rho_l U_\infty^2 / 2) t_\infty}$$

$$\dot{m}^- = \frac{C_{dest} \rho_v \alpha_l \min[0, p - p_v]}{(\rho_l U_\infty^2 / 2) t_\infty}$$

$$\dot{m}^+ = \frac{C_{prod} \rho_v (1 - \alpha_l) \max[0, p - p_v]}{(\rho_l U_\infty^2 / 2) t_\infty}$$

$$\dot{m}^+ = \frac{C_{prod} \rho_v \alpha_l^2 (1 - \alpha_l)}{t_\infty}$$

Option #3 : Singhal et al. 2002

Option #5 : Merkle et al. 2006

$$\dot{m}^- = c_{dest} \frac{\sqrt{k}}{T} \rho_L \rho_v Y_L \frac{dR}{dt}$$

$$\dot{m}^- = -k_v \frac{\rho_v \alpha_l}{t_\infty} \min \left\{ 1, \max \left(\frac{p_v - p}{k_p P_v}, 0 \right) \right\}$$

$$\dot{m}^+ = c_{prod} \frac{\sqrt{k}}{T} \rho_L \rho_v Y_V \frac{dR}{dt}$$

$$\dot{m}^+ = k_l \frac{\rho_v \alpha_v}{t_\infty} \min \left\{ 1, \max \left(\frac{p - p_v}{k_p P_v}, 0 \right) \right\}$$

Option #4 : Yuan et al. 2003

$$\dot{m}^- = \begin{cases} -p_v L (4\pi N)^{1/3} (3\alpha_v)^{2/3} \sqrt{\frac{2(p_v - p)}{\rho_l}} & \text{if } p \leq p_v \text{ and } \alpha_v < 1 \\ 0 & \text{if } p > p_v \text{ or } \alpha_v = 1 \end{cases}$$

$$\dot{m}^+ = \begin{cases} p_v L (4\pi N)^{1/3} (3\alpha_v)^{2/3} \sqrt{\frac{2(p - p_v)}{\rho_l}} & \text{if } p \geq p_v \\ 0 & \text{if } p < p_v \end{cases}$$

Boiling Model

Lee's boiling model

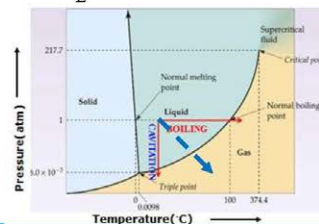
$$\dot{m}_T^+ = C_{ev} \rho_v \alpha_v \max \left(\frac{T_v - T}{T_v}, 0 \right)$$

$$\dot{m}_T^- = C_{it} \rho_l \alpha_l \min \left(\frac{T_v - T}{T_v}, 0 \right)$$

PNU boiling model (based Merkle model), 2014

$$\dot{m}_T^- = \frac{C_{Tdest} \rho_l \alpha_l}{\frac{1}{2} \rho_\infty U_\infty t_\infty} \min(T_{sat} - T, 0)$$

$$\dot{m}_T^+ = \frac{C_{Tprod} \rho_v \alpha_v}{\frac{1}{2} \rho_\infty U_\infty t_\infty} \max(T_{sat} - T, 0)$$

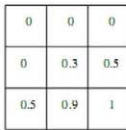


Numerical Methods

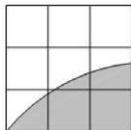
VOF MODEL + Multi phase Code

$$\frac{\partial \alpha}{\partial t} + \frac{\partial}{\partial x}(u\alpha) + \frac{\partial}{\partial y}(v\alpha) = 0$$

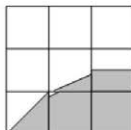
$$\frac{1}{J} \frac{\partial \alpha}{\partial t} + \frac{\partial}{\partial \xi} \left(\frac{1}{J} U \alpha \right) + \frac{\partial}{\partial \eta} \left(\frac{1}{J} V \alpha \right) = 0$$



(a) Volume Fractions



(b) True Interface



(c) Piecewise Linear Approximation

* AHEM : Advanced Homogeneous Mixture

AHEM Mixture MODEL + Multi phase Code

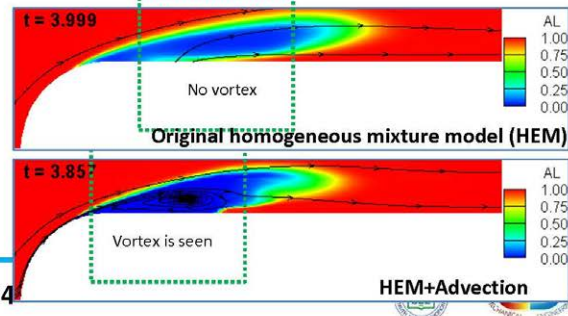
Adding the solution of volume fraction advection equations for both vapor and gaseous mixture to the solution of HEM

+ For vapor phase

$$\frac{\partial \alpha_v}{\partial t} + U \frac{\partial \alpha_v}{\partial \xi} + V \frac{\partial \alpha_v}{\partial \eta} + W \frac{\partial \alpha_v}{\partial \zeta} = - \frac{(\dot{m}^+ + \dot{m}^-)}{\rho_v}$$

+ For non-condensable gaseous mixture

$$\frac{\partial \alpha_g}{\partial t} + U \frac{\partial \alpha_g}{\partial \xi} + V \frac{\partial \alpha_g}{\partial \eta} + W \frac{\partial \alpha_g}{\partial \zeta} = 0$$



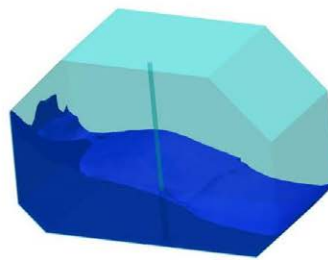
Numerical Methods

VOF equation in Cartesian coordinates

$$\frac{\partial \alpha}{\partial t} + \frac{\partial}{\partial x}(u\alpha) + \frac{\partial}{\partial y}(v\alpha) = 0$$

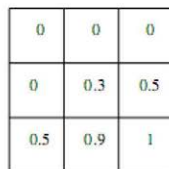
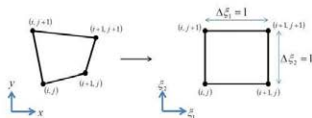
Transformation to curvilinear coordinates

$$\frac{1}{J} \frac{\partial \alpha}{\partial t} + \frac{\partial}{\partial \xi} \left(\frac{1}{J} U \alpha \right) + \frac{\partial}{\partial \eta} \left(\frac{1}{J} V \alpha \right) = 0$$

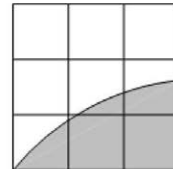


The contravariant velocities:

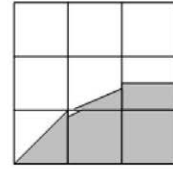
$$U = \xi_x u + \xi_y v; \quad V = \eta_x u + \eta_y v$$



(a) Volume Fractions



(b) True Interface



(c) Piecewise Linear Approximation



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Results and Discussion

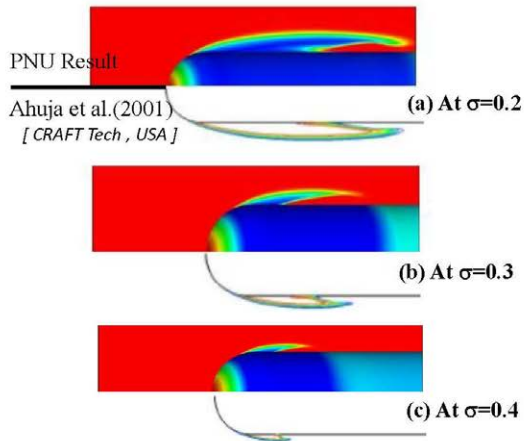
- Natural & Ventilated(Artificial) Caviation

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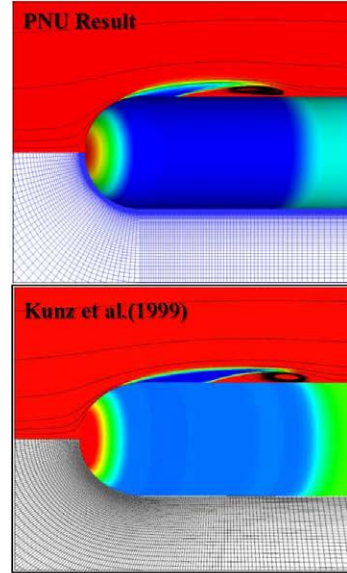


Natural Cavitation

Comparison with Ahuja's Result

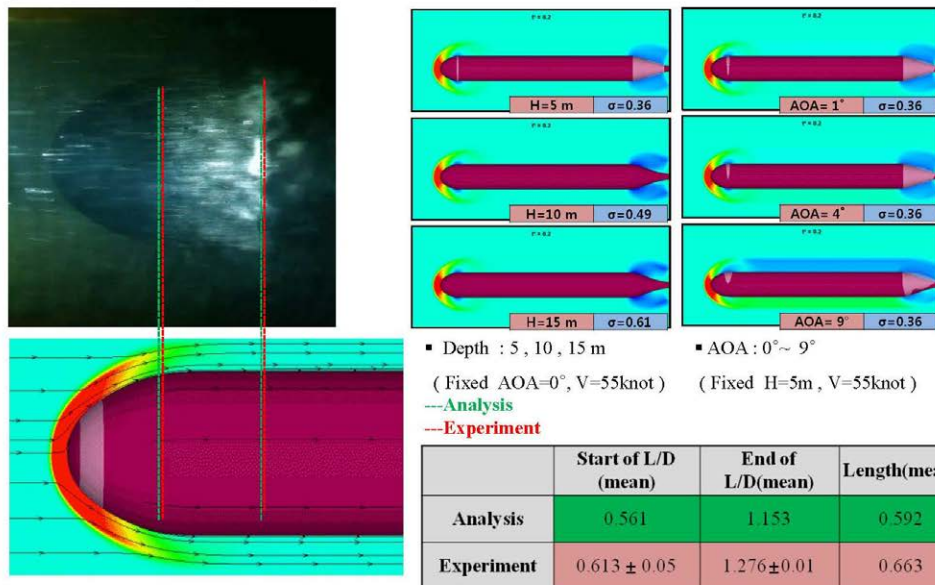


Comparison with Kunz's Result(ONR/ARL)



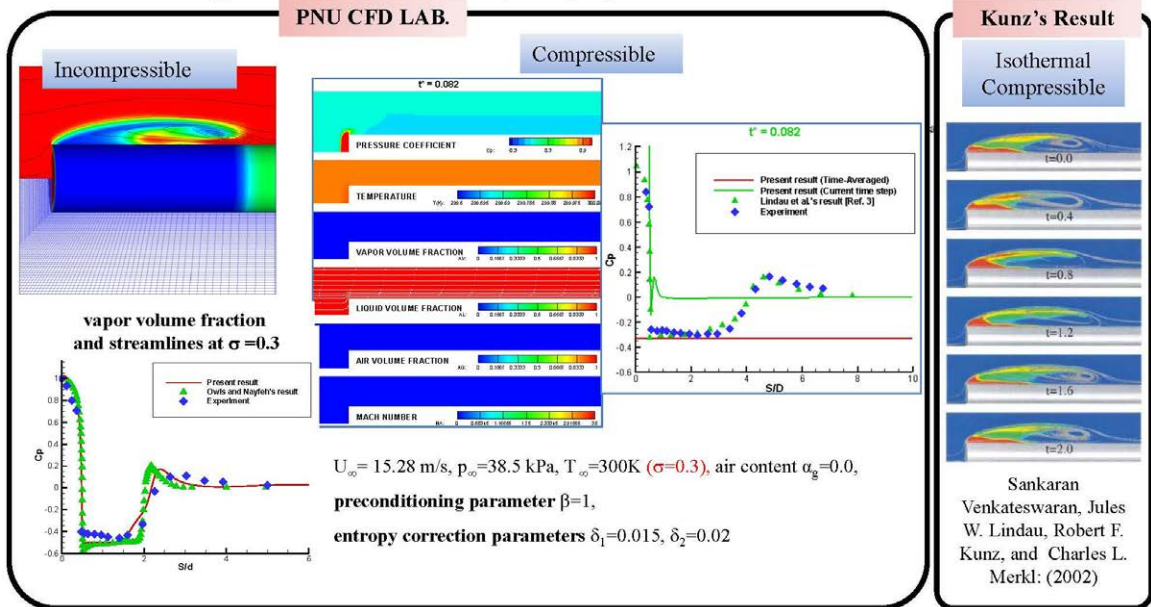
Natural Cavitation

Comparison with Experiment (ADD, Korea)



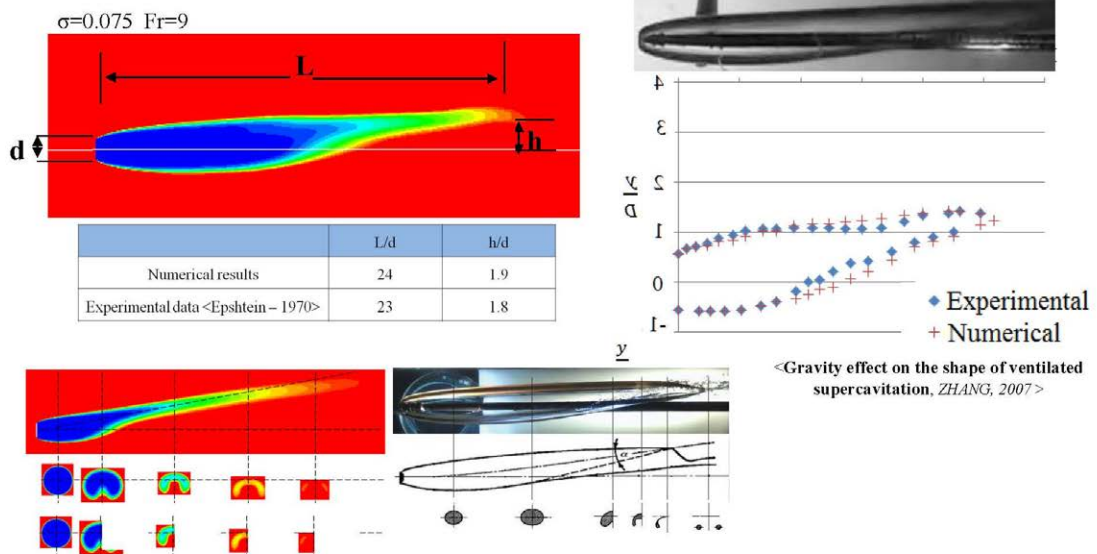
Compressible Effects

Cavitating Flows over a 0 Caliber(blunt) Cylinder



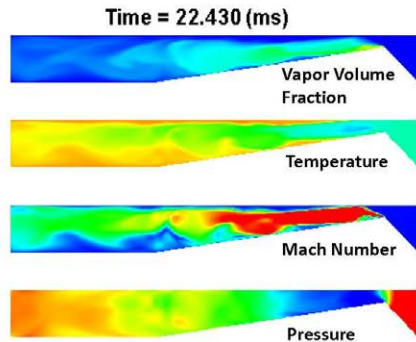
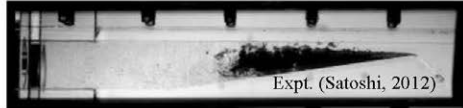
Buoyancy (Gravity) Effect

Disk with Gravity Effect : Ventilated Cavitation

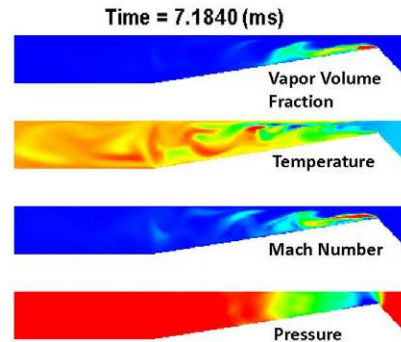


Nozzle Cavitation

Cavitating Flows in a Nozzle



Flow conditions:
 $U_{inlet} = 3.5 \text{ m/s}$; $T_{inlet} = 27^\circ\text{C}$; $p_{outlet} = 94.5 \text{ kPa}$
 $(\sigma_{outlet} \approx 14.9)$



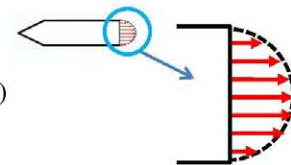
Flow conditions:
 $U_{inlet} = 3.5 \text{ m/s}$; $T_{inlet} = 27^\circ\text{C}$; $p_{outlet} = 124 \text{ kPa}$
 $(\sigma_{outlet} \approx 19.8)$



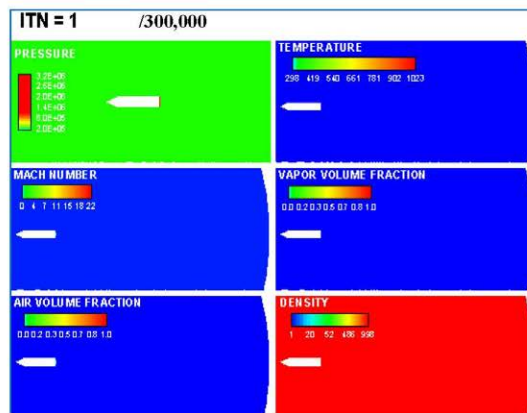
Compressible Effects

Ventilated Cavitation with Temperature Difference

- Free stream conditions: $U_\infty = 112 \text{ m/s}$
 $p_\infty = 101.325 \text{ Pa}$; $T_\infty = 25^\circ\text{C}$ (298.15K) ($M_\infty = 0.075$)
- Conditions at the blowhole: $U_{ven} = 392 \text{ m/s}$,
 $p_{ven} = 1.2 \times 10^6 \text{ Pa}$; $T_{ven} = 750^\circ\text{C}$ (1023.15K)



→ : Injecting Gas



Steady equations were solved



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Supercavitating Torpedo

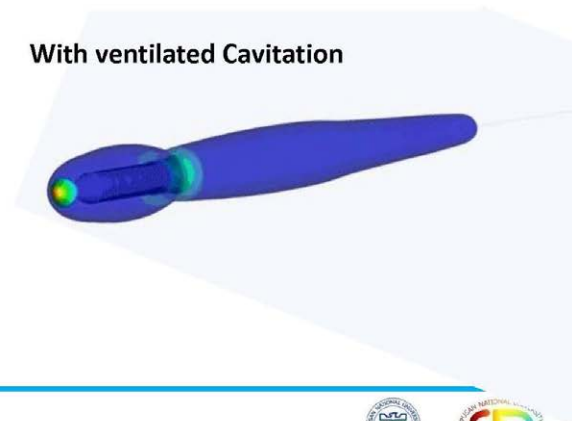
Barracuda



Without ventilated Cavitation



With ventilated Cavitation



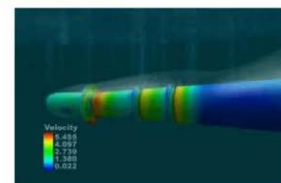
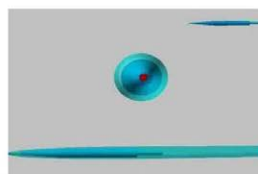
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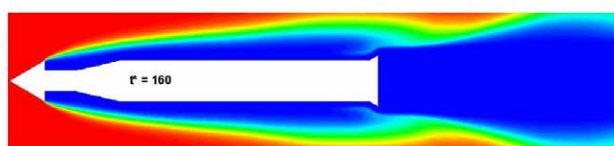
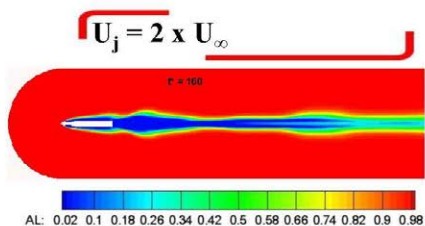
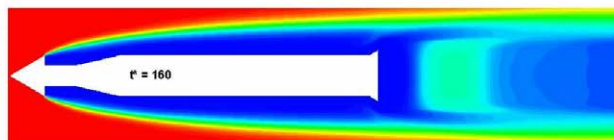
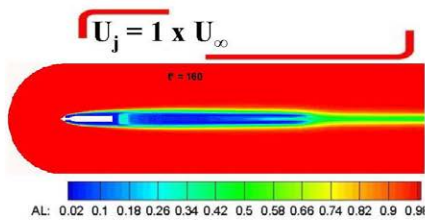
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2019.04.23

Ventilated Cavitation

Cavity Instability



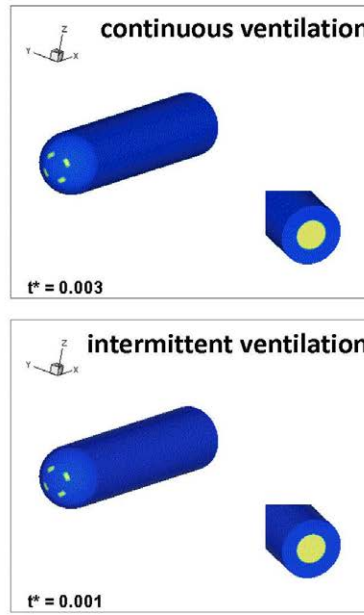
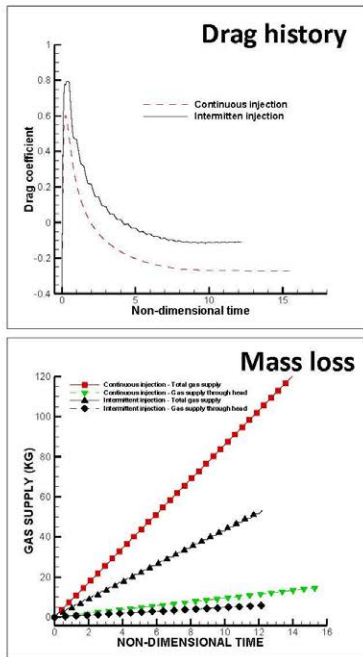
< ONR / ARL >



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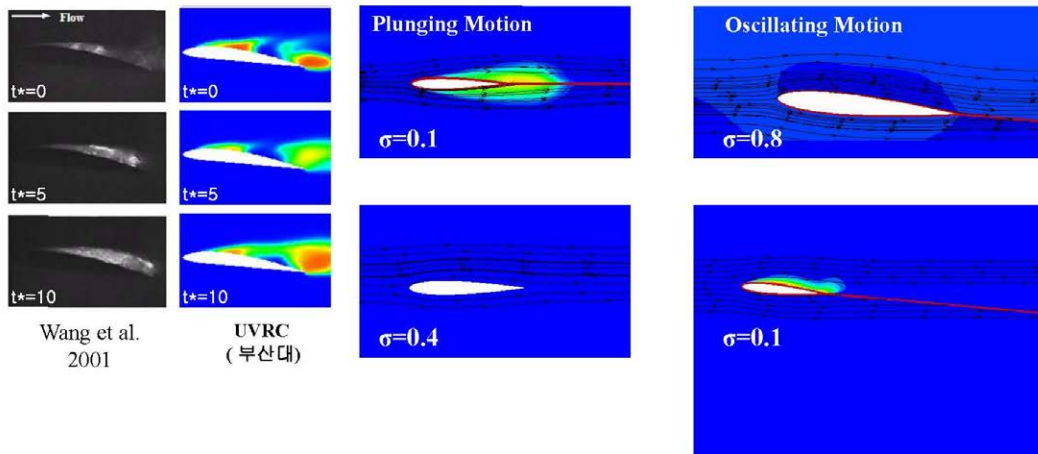
Ventilated Cavitation - continuous & intermittent ventilation



< Surface pressure and gas contour >

Cavitation over Hydrofoil

Cavitating Flows over Hydrofoil

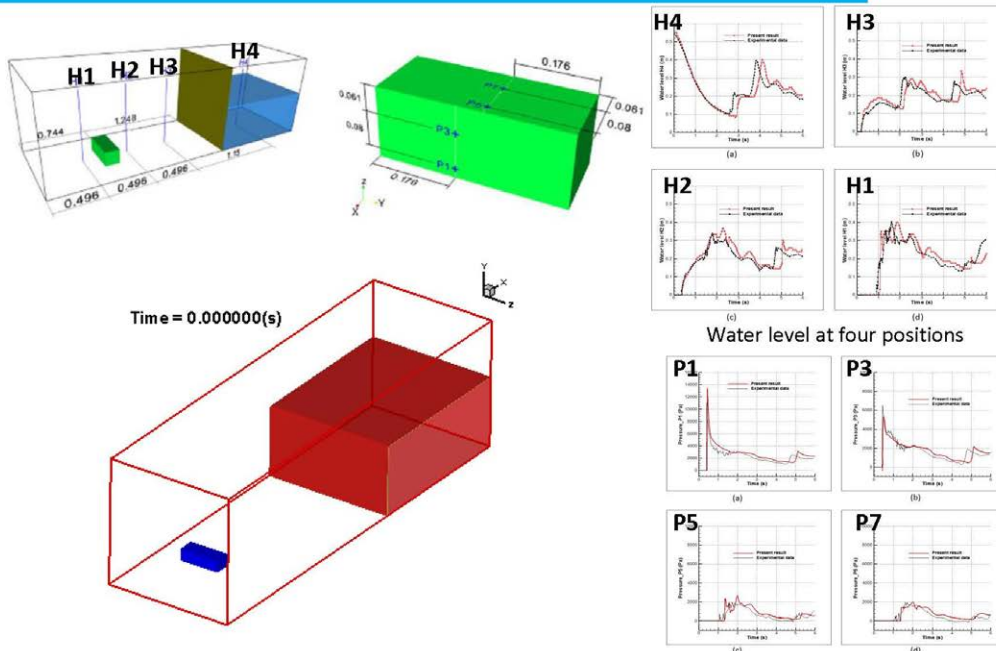


Results and Discussion

- Free Surface Simulations



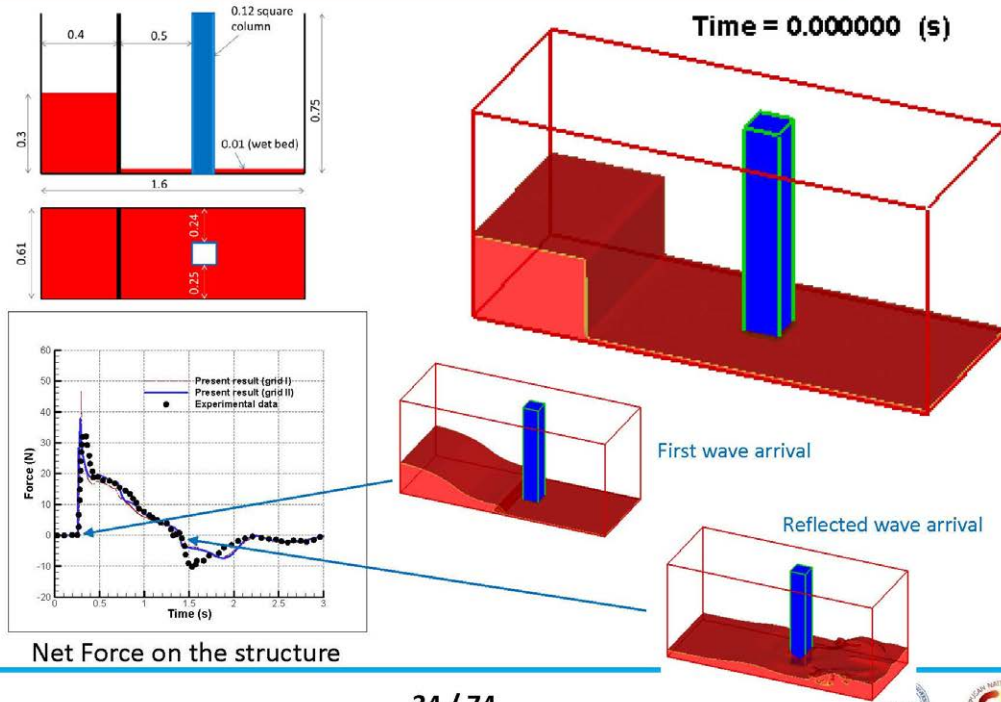
Dam-break problem with an object



Pressures at four gauges



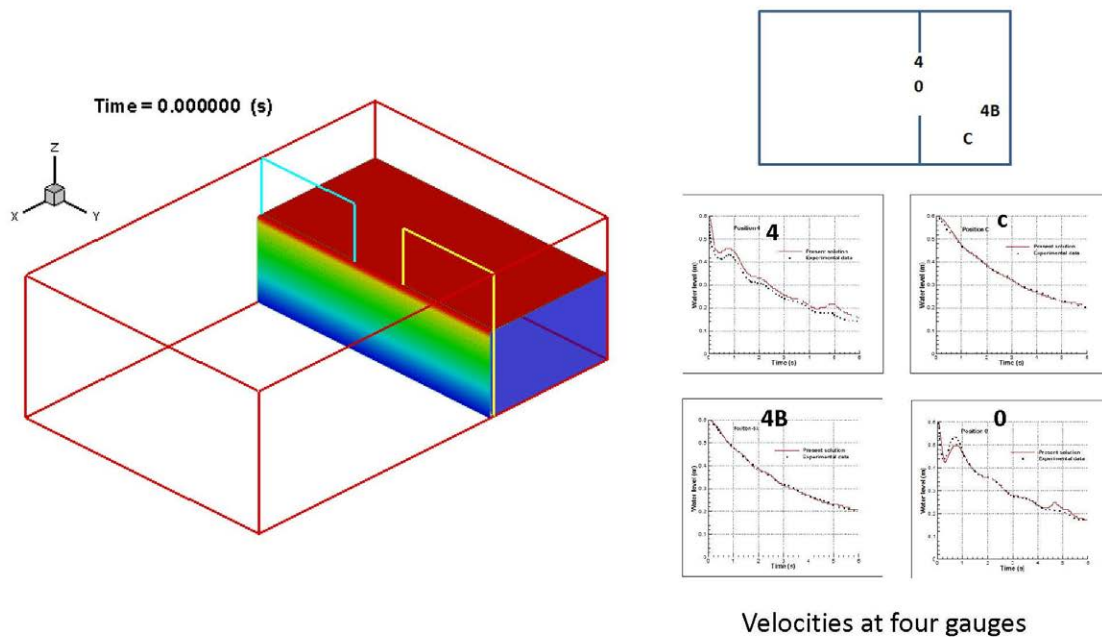
Dam-break problem with an object



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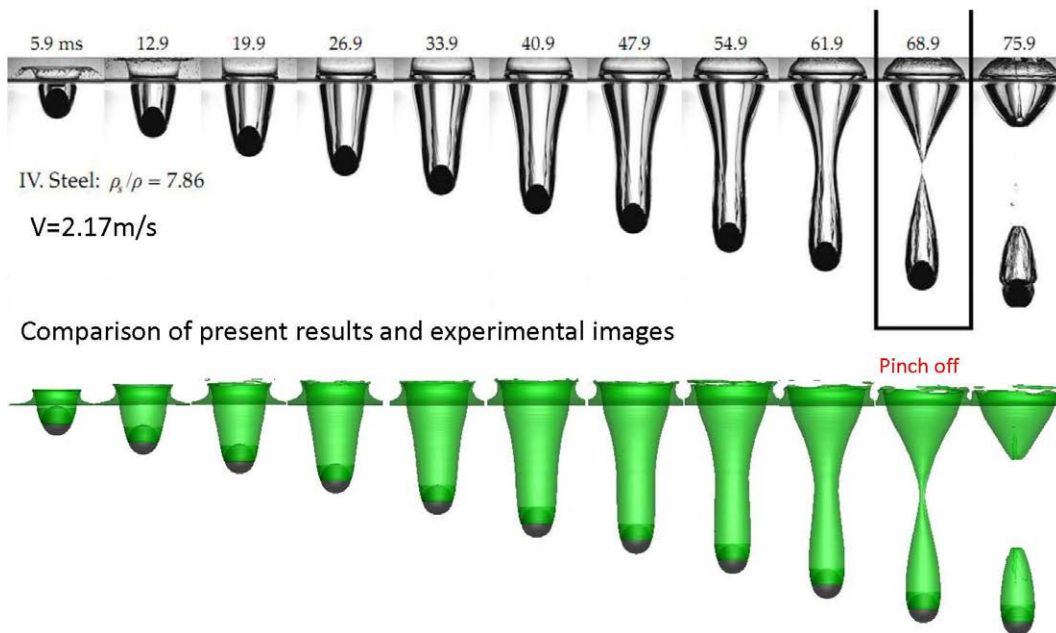
Dam-break problem



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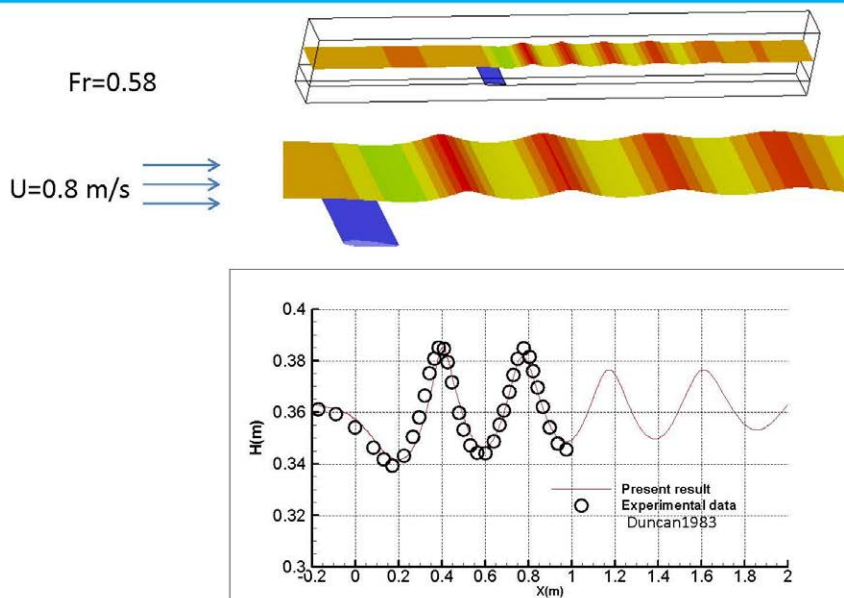
Simulation of a sphere entering into water using Chimera technique



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Steady Free Surface Flows about hydrofoil NACA0012

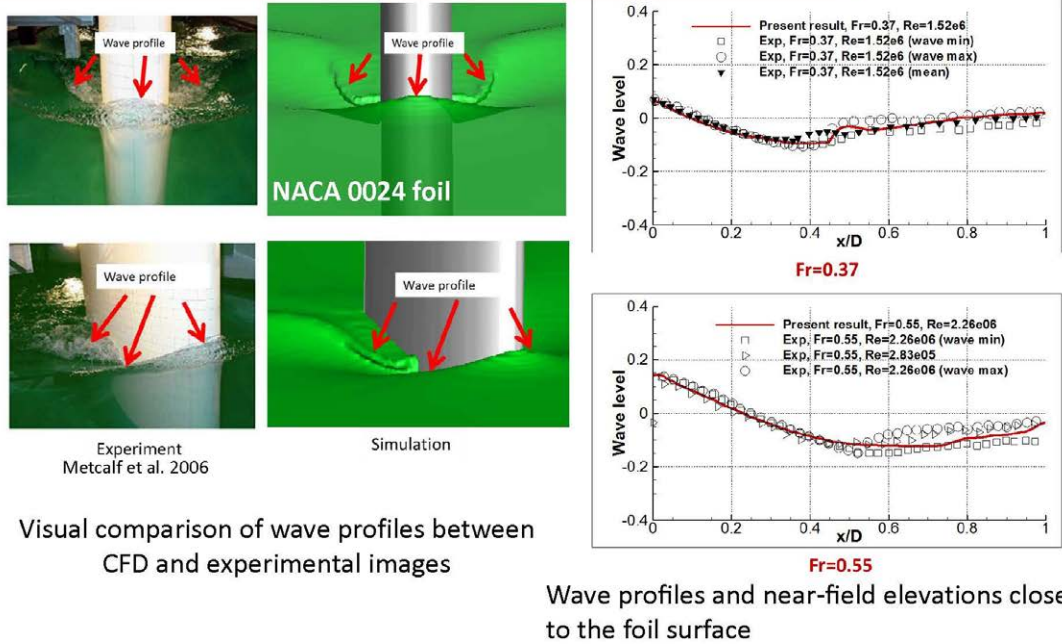


Comparison of wave profile between numerical and experimental results

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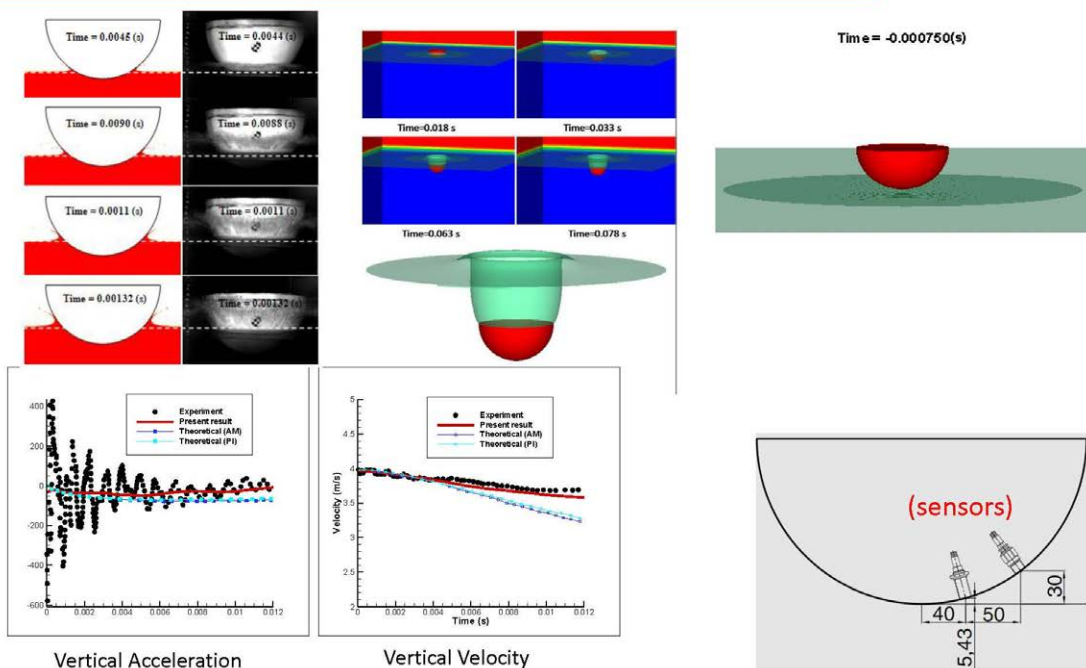
Free-surface Wave



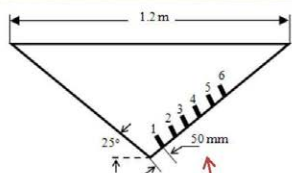
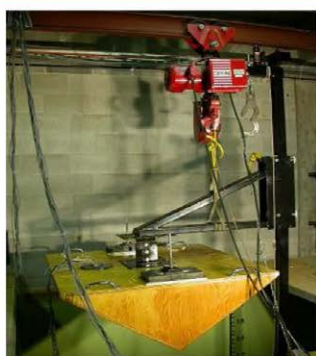
Results and Discussion

- Water Entry Problems

Water Impact Force Analysis

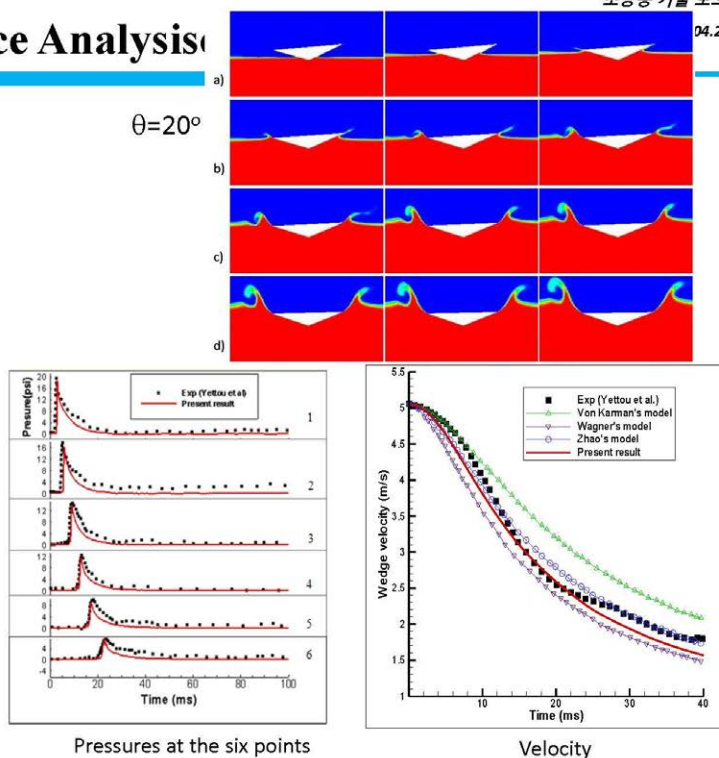


Water Impact Force Analysis



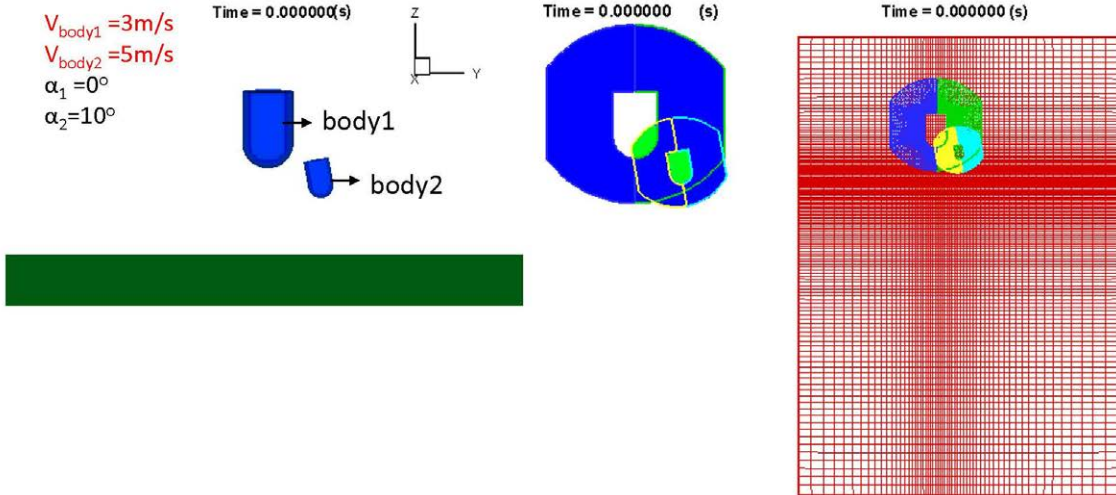
Six Pressure Transducers
E.M. Yettou et al. 2006

$\theta = 20^\circ$



6 DOF Motion Analysis and Chimera Grid

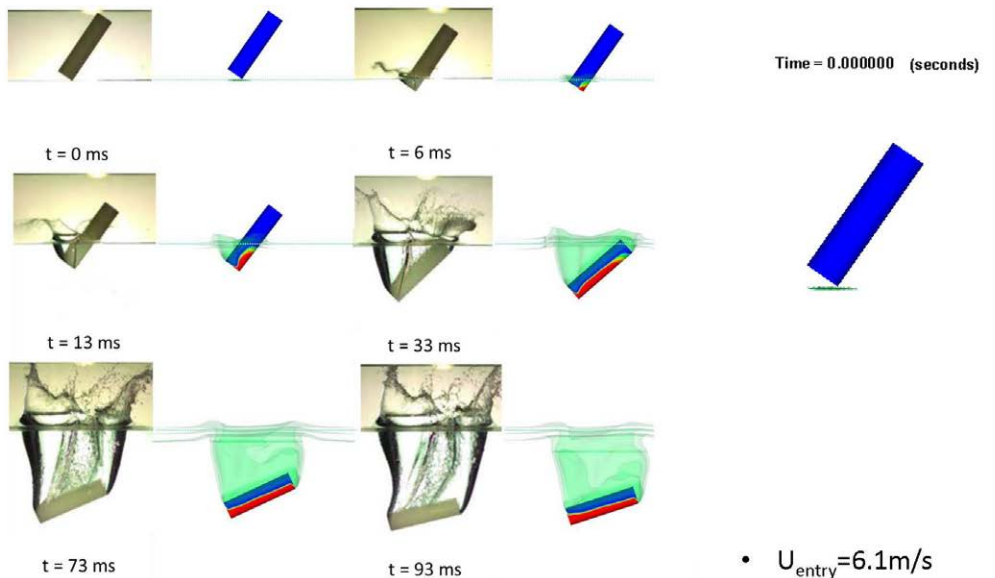
Multi-body entering to the water with multiple Chimera grid



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Simulation of a water entry cylinder with inclined angle

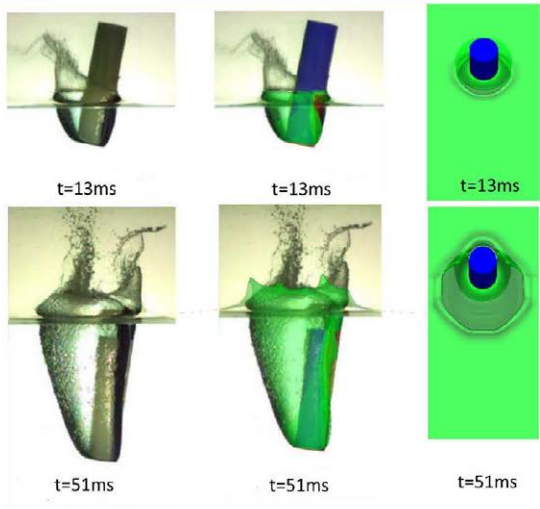


***'Evolution of three-dimensional cavitation following water entry of an inclined cylinder', Zhaoyu Wei, et.al(2012)

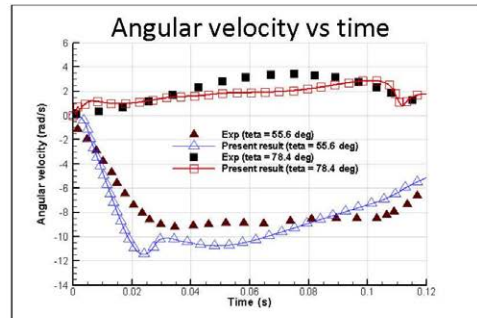
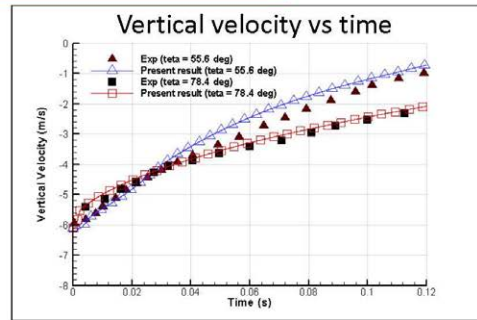
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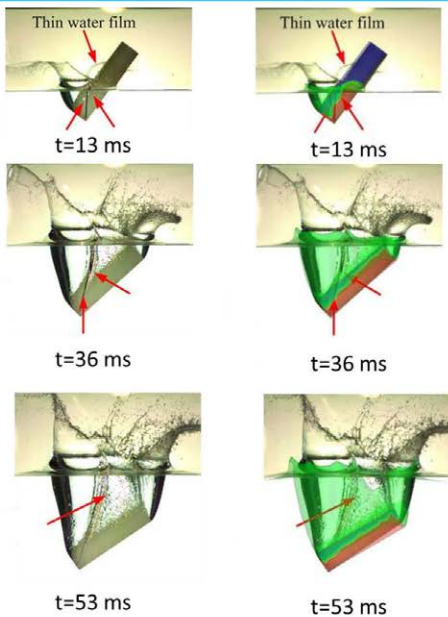
Simulation of a water entry cylinder with inclined angle



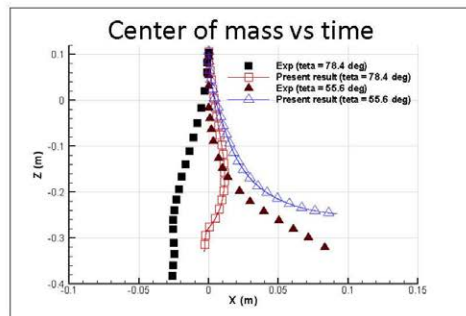
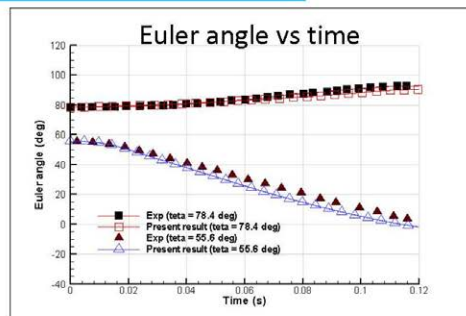
Snapshots of water entry cylinder at inclined angle of **78.4 degrees**



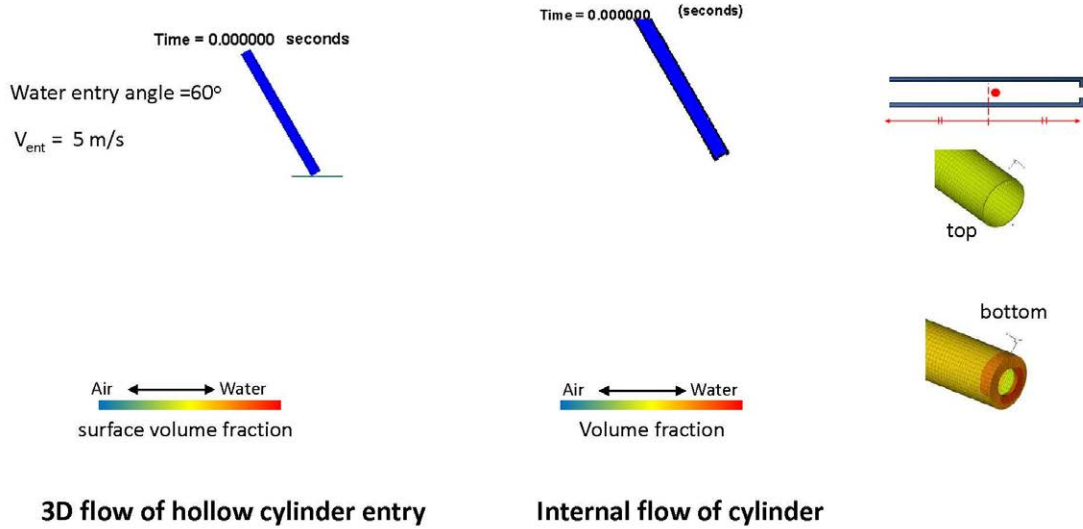
Simulation of a water entry cylinder with inclined angle



Snapshots of water entry cylinder at inclined angle of **53 degrees**



6DOF water entry behavior of hollow cylinder



6DOF water entry behavior of solid/hollow cylinder

- Inclined angles: 60°
- initial entry velocities: 10 m/s



Model 1. 
Solid cylinder

Model 2. 
Hollow cylinder

Ricochet problem

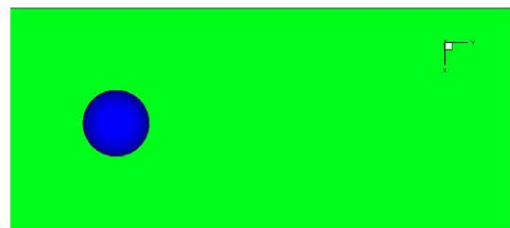
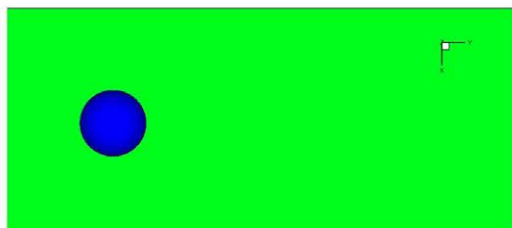
Case1: 10m/s

Case2: 5m/s

Time = 0.000000



Time = 0.000000



Diameter of sphere = 0.025 m
Density of sphere = water density



Results and Discussion

*- Full Fluids Equation Solver
for Multiphase Flow Analysis*



Full Fluids Equations

2D Shock Bubble Interaction

Air/Heliumbubble system

Air $\left\{ \begin{array}{l} \rho_a = 1.29 \text{ kg/m}^3, u_a = 0 \text{ m/s}, p_a = 10^5 \text{ Pa} \\ \gamma_a = 1.4, \pi_a = 0 \text{ Pa} \end{array} \right.$

Helium

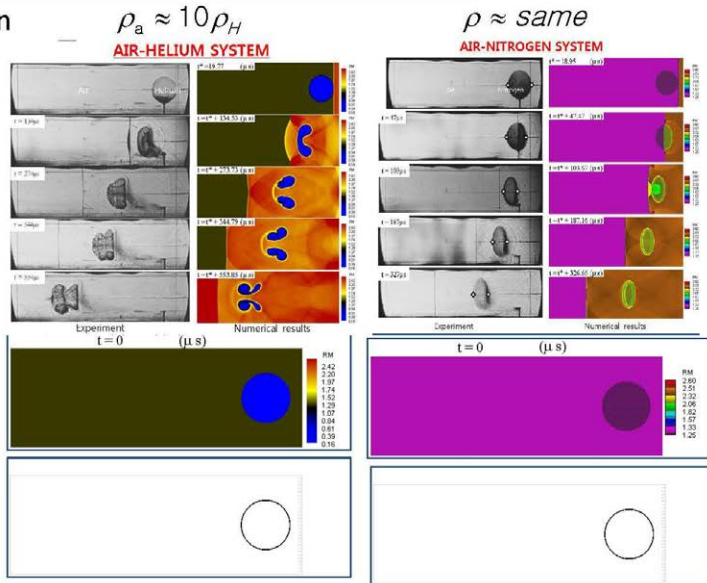
$\left\{ \begin{array}{l} \rho_H = 0.167, u_H = 0, p_H = 10^5 \text{ Pa} \\ \gamma_H = 1.67, \pi_H = 0 \text{ Pa} \end{array} \right.$

Air/Nitrogenbubble system

Air $\left\{ \begin{array}{l} \rho_a = 1.29 \text{ kg/m}^3, u_a = 0 \text{ m/s}, p_a = 10^5 \text{ Pa} \\ \gamma_a = 1.4, \pi_a = 0 \text{ Pa} \end{array} \right.$

Nitrogen

$\left\{ \begin{array}{l} \rho_N = 1.25 \text{ kg/m}^3, u_N = 0, p_N = 10^5 \text{ Pa} \\ \gamma_N = 1.4, \pi_N = 0 \text{ Pa} \end{array} \right.$



Second order HLL scheme with MUSCL interpolation ($\beta=1$)



SHOCK BUBBLE INTERACTIONS

AIR-HELIUM SYSTEM

$(\rho_{air} / \rho_{Helium} = 7.72)$

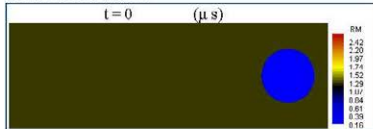
AIR-NITROGEN SYSTEM

$(\rho_{air} / \rho_{Nitrogen} = 1.03)$

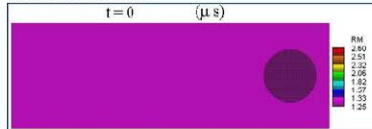
AIR- KRYPTON SYSTEM

$(\rho_{air} / \rho_{Krypton} = 0.37)$

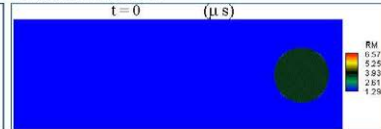
MIXTURE DENSITY



MIXTURE DENSITY



MIXTURE DENSITY



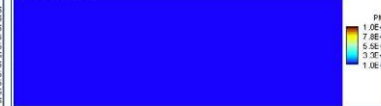
PRESURE (Pa)



PRESURE (Pa)



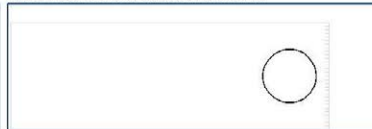
PRESURE (Pa)



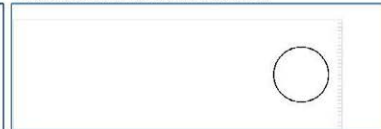
VECTOR FIELD & BUBBLE SHAPE



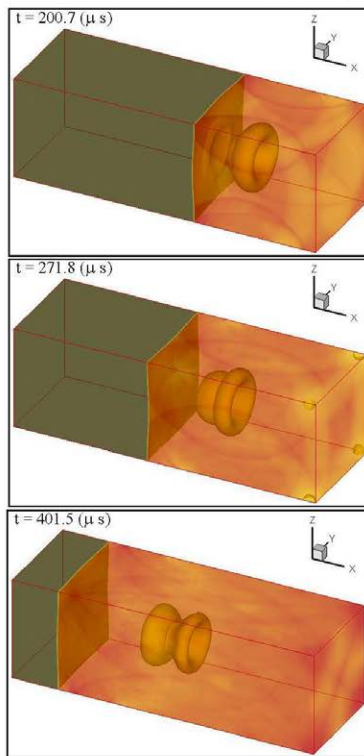
VECTOR FIELD & BUBBLE SHAPE



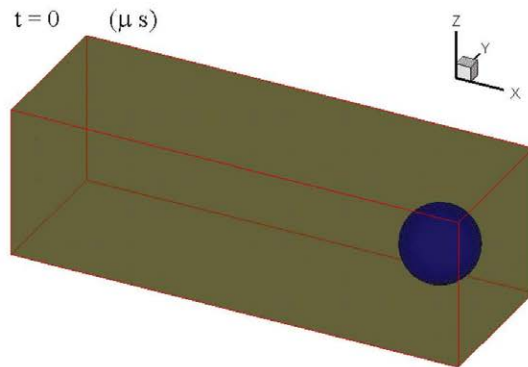
VECTOR FIELD & BUBBLE SHAPE



THREE-DIMENSIONAL SHOCK BUBBLE INTERACTION



AIR-HELIUM SYSTEM



Full Fluids Equations

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Richtmeyer-Meshkov Instability

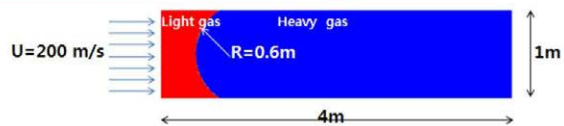
Initial conditions :

$$\begin{cases} \rho_{\text{lightgas}} = 1 \text{ kg/m}^3, & p_{\text{lightgas}} = 10^5, \\ u_{\text{lightgas}} = 0 \text{ m/s}, & v_{\text{lightgas}} = 0 \text{ m/s} \end{cases}$$

$$\begin{cases} \rho_{\text{heavygas}} = 50 \text{ kg/m}^3, & p_{\text{heavygas}} = 10^5 \text{ Pa}, \\ u_{\text{heavygas}} = 0 \text{ m/s}, & v_{\text{heavygas}} = 0 \text{ m/s}, \end{cases}$$

$$\gamma_{\text{lightgas}} = 1.4, \quad \pi_{\text{lightgas}, \infty} = 0 \text{ (Pa)},$$

$$\gamma_{\text{heavygas}} = 1.6, \quad \pi_{\text{heavygas}, \infty} = 0 \text{ (Pa)}$$

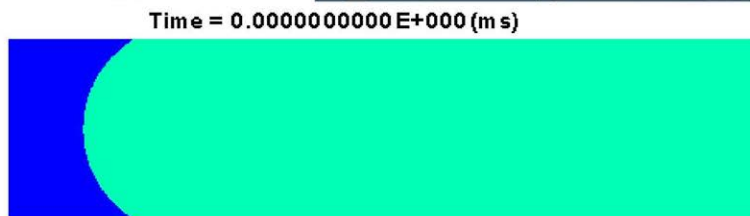
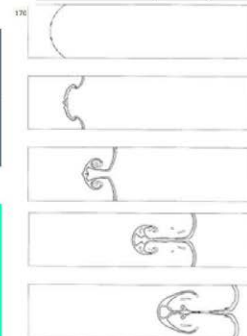


Saurel et. al (2009)

S. Saurel et al. Journal of Computational Physics 228 (2009) 1074–1112



Saurel et. al (1999)



RM 10.00 13.00 15.39 40.00 70.00 100.00 130.00 160.00 190.00 220.00 250.00

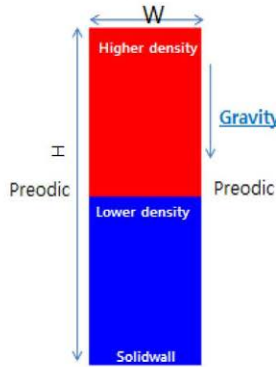
Second order HLL scheme with MUSCL interpolation ($\beta=1$)



Full Fluids Equations

Rayleigh-Taylor Instability

Kelvin-Helmholtz instability may occur :
 - If a fluid of high density resting on top of a low-density fluid
 - And if a small disturbance, such as a wave, is introduced at the interface



Initial conditions :

$$\begin{cases} \rho_1 = 2\text{kg/m}^3, & u_1 = 0 \text{ m/s}, \\ v_1 = 0 \text{ m/s} \end{cases}$$

$$\begin{cases} \rho_2 = 1\text{kg/m}^3, & u_2 = 0 \text{ m/s}, \\ v_2 = 0 \text{ m/s}, \end{cases}$$

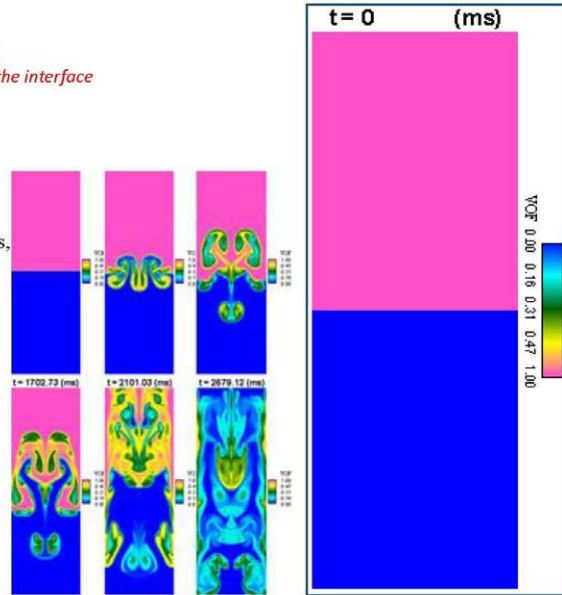
SG - EOS

$$\gamma_1 = 1.4, \quad p_{1,\infty} = 0 \text{ (Pa)},$$

$$\gamma_2 = 1.4, \quad p_{2,\infty} = 0 \text{ (Pa)}$$

The interface is initially perturbed with

$$v_{\text{dis}} = 0.05 \left[1 + \cos\left(\frac{2\pi x}{W}\right) \right] \left[1 + \cos\left(\frac{2\pi z}{H}\right) \right]$$



Full Fluids Equations

Kelvin-Helmholtz Instability

Initial conditions :

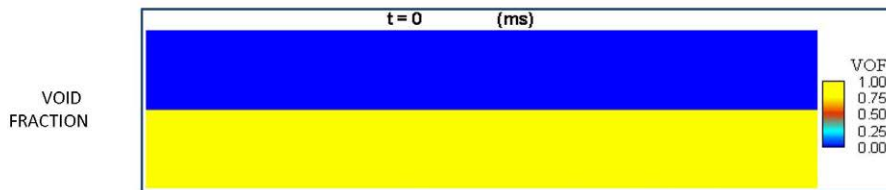
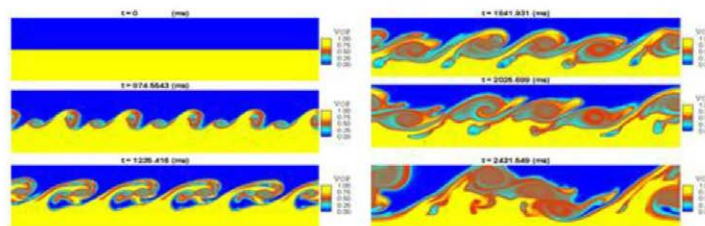
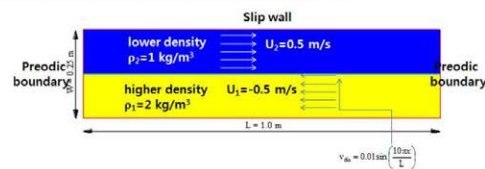
$$\begin{cases} \rho_1 = 2\text{kg/m}^3, & u_1 = 0 \text{ m/s}, \\ v_1 = 0 \text{ m/s} \end{cases}$$

$$\begin{cases} \rho_2 = 1\text{kg/m}^3, & u_2 = 0 \text{ m/s}, \\ v_2 = 0 \text{ m/s}, \end{cases}$$

SG - EOS

$$\gamma_1 = 1.4, \quad p_{1,\infty} = 0 \text{ (Pa)},$$

$$\gamma_2 = 1.4, \quad p_{2,\infty} = 0 \text{ (Pa)}$$



Expansion of Spherical Shock

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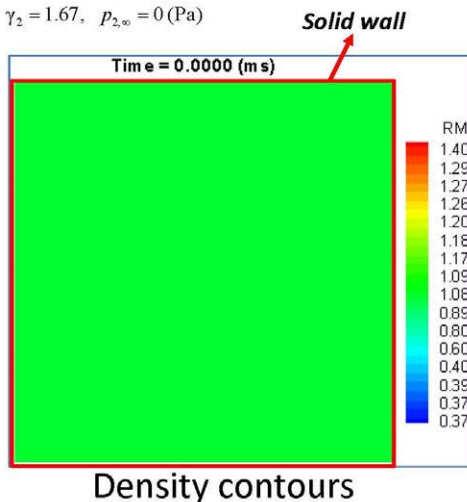
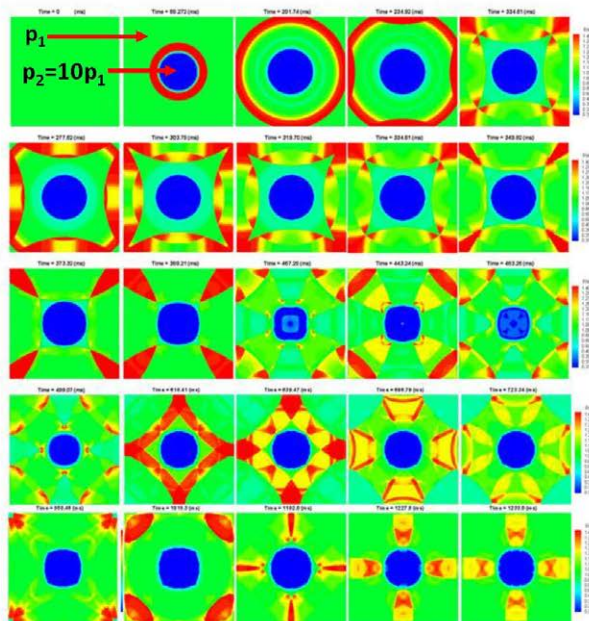
Initial conditions :

$$\begin{cases} \rho_1 = 1.0 \text{ kg/m}^3, & p_1 = 1 \text{ Pa}, & u_1 = 0 \text{ m/s}, \\ v_1 = 0 \text{ m/s} \\ \rho_2 = 1.0 \text{ kg/m}^3, & p_2 = 10 \text{ Pa}, & u_2 = 0 \text{ m/s}, \\ v_2 = 0 \text{ m/s}, \end{cases}$$

SG - EOS

$$\gamma_1 = 1.67, \quad p_{1,\infty} = 0 \text{ (Pa)},$$

$$\gamma_2 = 1.67, \quad p_{2,\infty} = 0 \text{ (Pa)}$$

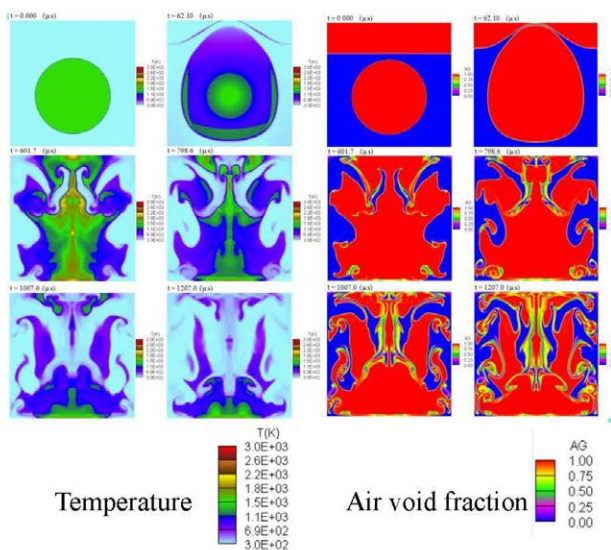
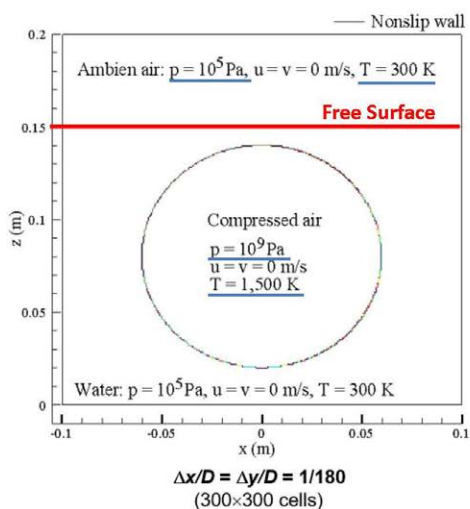


Underwater Explosion

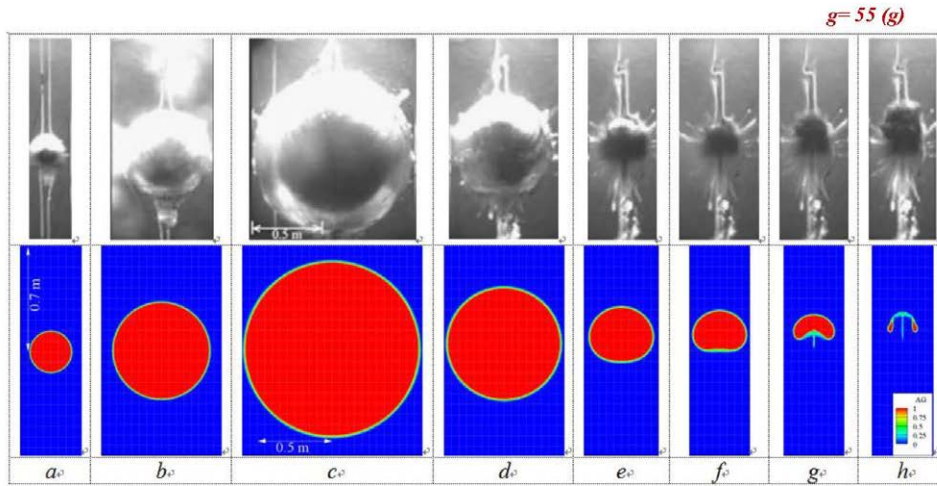
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2019.04.23

To examine the capability for capturing the interface with large jump in pressure, temperature, and density.



Underwater Explosion



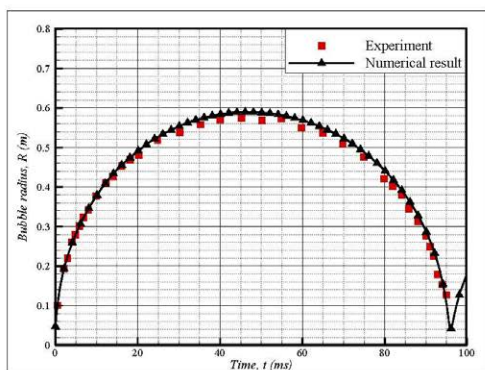
The evolution of bubble shape for case 55g; from the left to right the bubble shape at time, $t = 1, 7, 50, 85, 93, 94, 95,$ and 96 ms.

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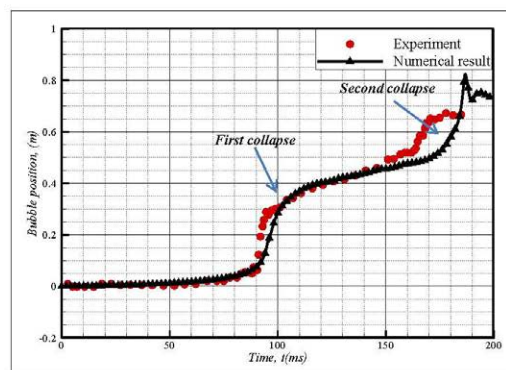


Underwater Explosion

$g = 55$ (g)



The comparison of bubble radius with experiment result for the 55 g case.



The position of the center mass of bubble as function of time

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Results and Discussion

- SuperCavitation

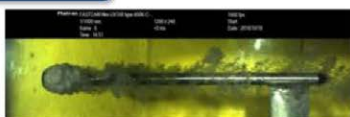


Super Cavitation

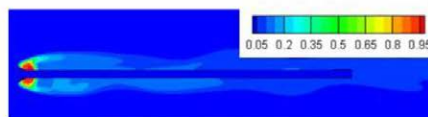
Ventilated flow

Fr=10.8

$$\frac{D_{H.KRISO}}{d_{C.KRISO}} = \frac{D_{H.PNU}}{d_{C.PNU}} = 9.49$$



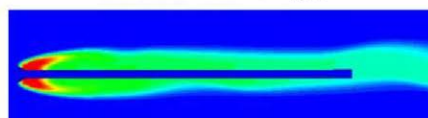
(a) Cq = 0.06



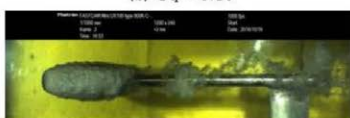
Simulation : Fr=10.8, Cq= 0.06



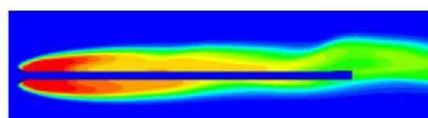
(b) Cq = 0.10



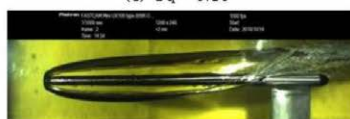
Simulation : Fr=10.8, Cq=0.1



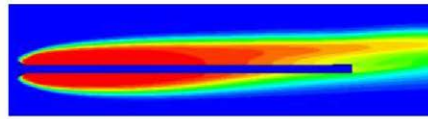
(c) Cq = 0.15



Simulation : Fr=10.8, Cq=0.15



(d) Cq = 0.2



Simulation : Fr=10.8, Cq=0.2

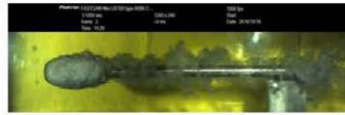


Super Cavitation

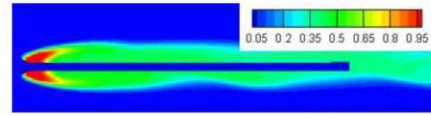
Ventilated flow

Fr=14.4

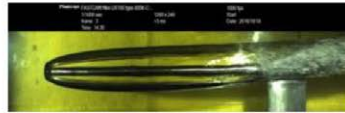
$$\frac{D_{H,KRISO}}{d_{C,KRISO}} = \frac{D_{H,PNU}}{d_{C,PNU}} = 9.49$$



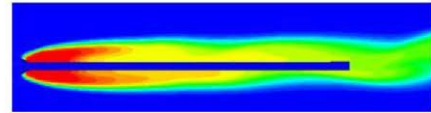
(b) Cq = 0.10



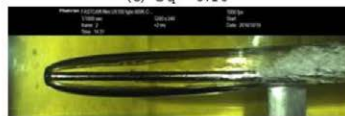
Simulation: Cq = 0.10



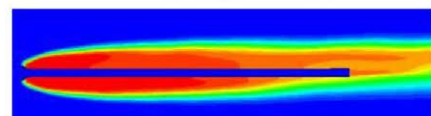
(c) Cq = 0.15



Simulation: Cq = 0.15



(d) Cq = 0.2



Simulation: Cq = 0.2

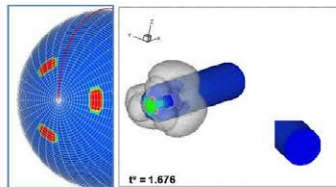


Super Cavitation

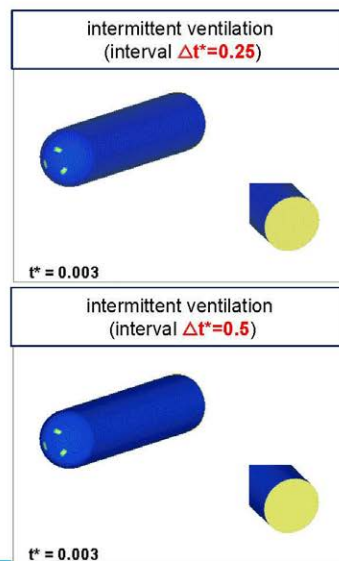
분사 간격 변화

* Flow conditions:
 $U_{\infty}=50$ m/s, $p_{\infty}=1.5 \times 10^5$ kPa, $T_{\infty}=15^{\circ}$ C

* Ventilation conditions
 $T_g=115^{\circ}$ C ; $Q_g=36$ kg/s ; $p_g=4.0 \times 10^6$ Pa



3 holes ; $A_{ven}=0.01648$ m²



Super Cavitation

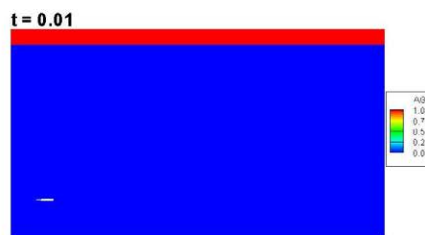
수면 변화

Inlet velocity = 5m/s; Inlet P= 1.00506kPa; Injection velocity = 2m/s;
Injection P = 4.0kPa; Re = 408,482.89; Fr = 17.06;

t = 0.01



10m below the free surface



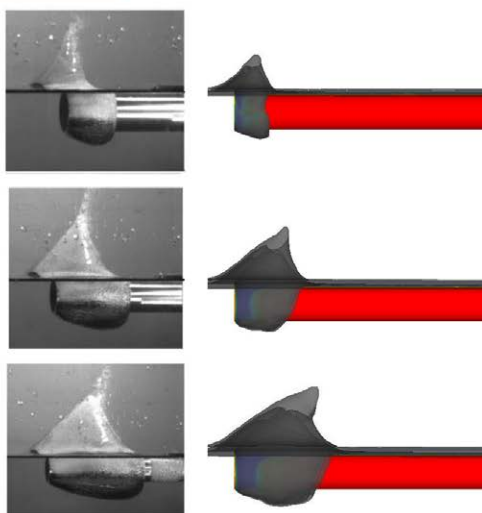
20m below the free surface



Super Cavitation

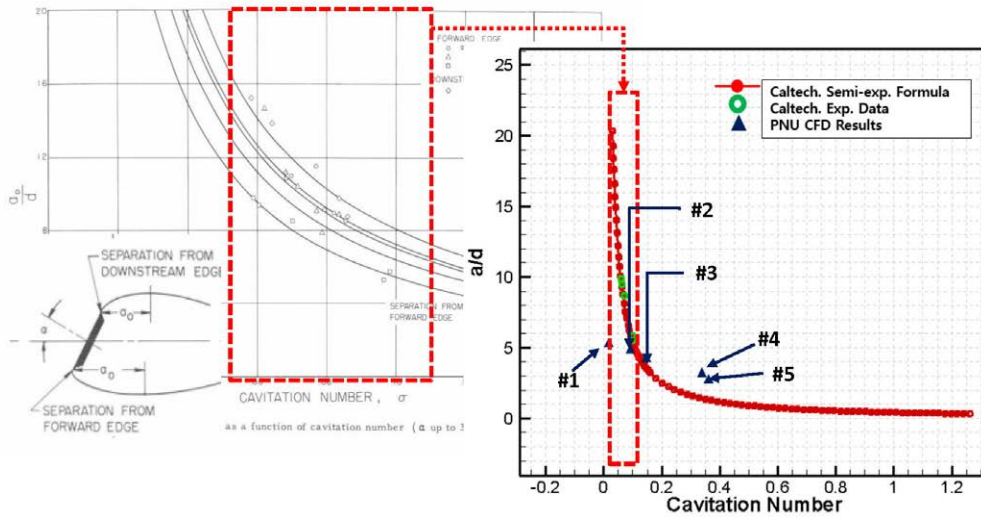
수면 변화

Velocity: U=19.1 m/s
Diameter: D=37mm
Cav number: 0.54



Super Cavitation

Cavitation Flow In Disk Cavitator with AoA



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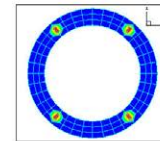
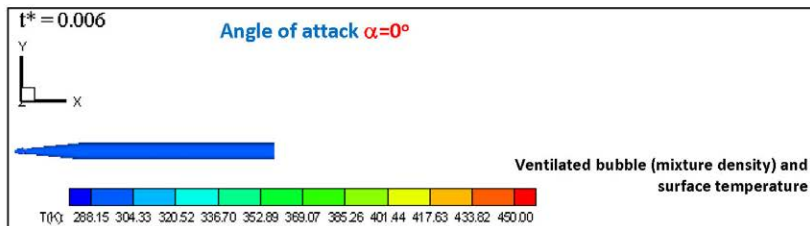


Super Cavitation

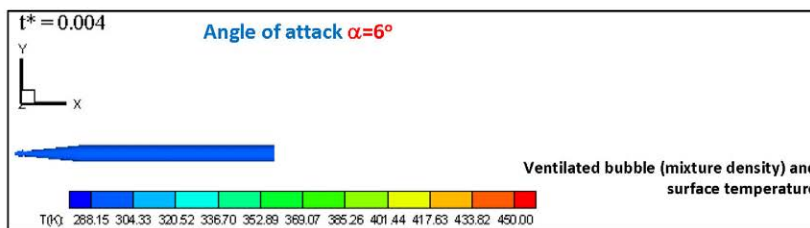
Cavitation Flow In Torpedo with AoA

Flow conditions $U_\infty=50$ m/s , $p_\infty = 101,325$ Pa, $T_\infty = 288.15$ K

Ventilation conditions $P_{ven} = 2.5$ MPa, $T_{ven} = 375$ K, $U_{ven}=390$ m/s



Four holes



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Super Cavitation

Cavitation Flow In Cavitator with AoA

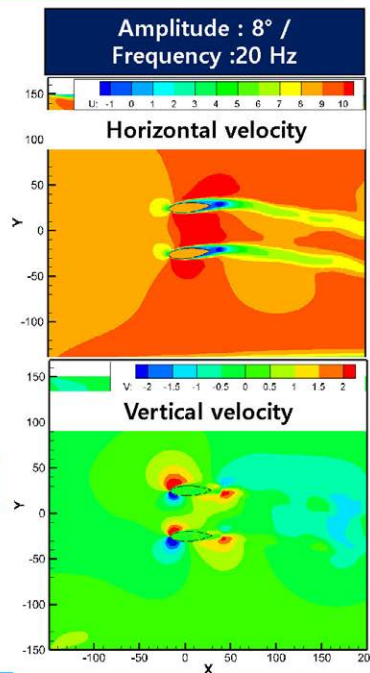
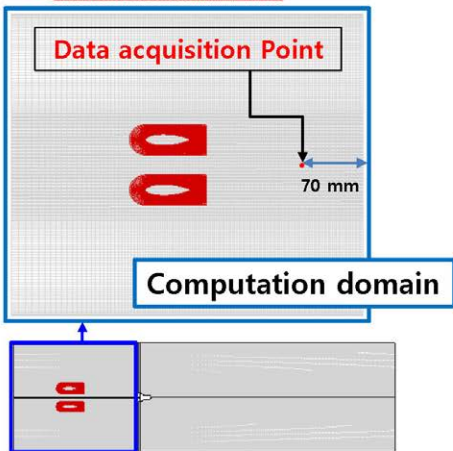
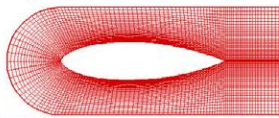


Cavitator AoA	Cavity shape (volume fraction)	Point of collapse	Wetted Surface ratio	Drag (cavitator only)
-10°		14d _c , 48d _c (Underside)	~5%	
0°		-	0.0	
+10°		14d _c , 52d _c (Upperside)	~5%	
+20°		11d _c (Upperside)	~20%	

Super Cavitation

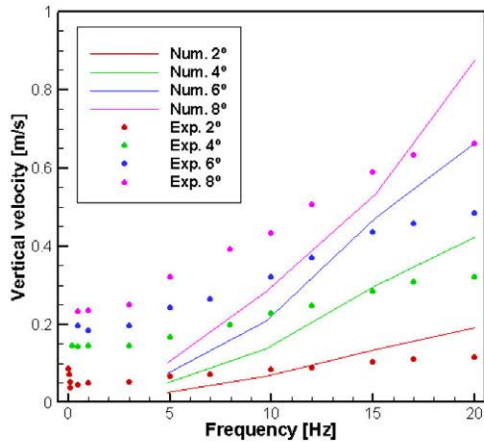
비정상 외란 해석

Hydrofoil (KH002, KRISO)

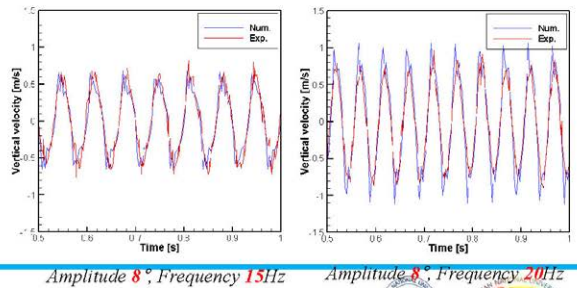
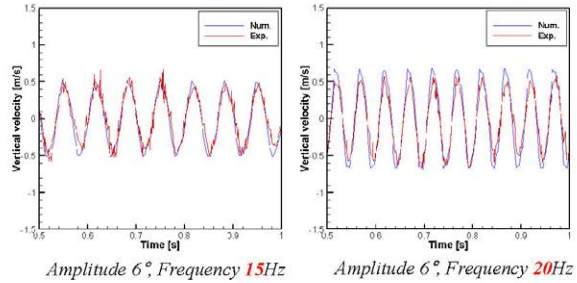


Super Cavitation

비정상 외란 해석



The maximum velocity in vertical direction according to amplitude and frequency

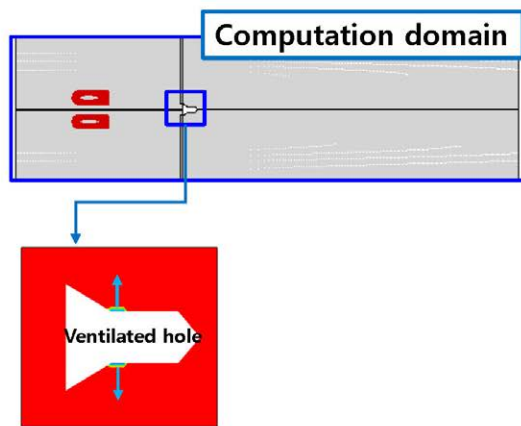


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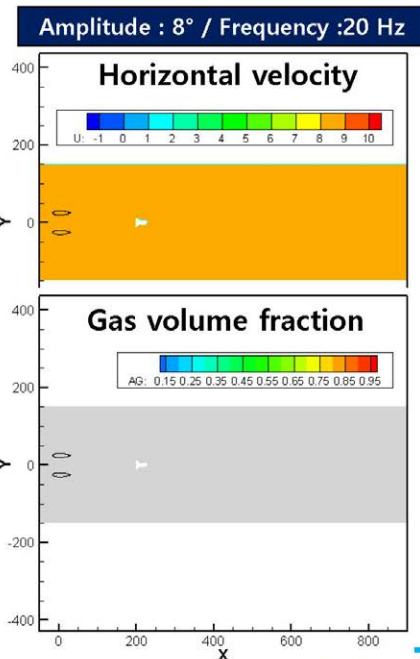


Super Cavitation

비정상 외란 해석



Cavitator with ventilated hole



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Contents

1. Introduction
2. Governing equations & Numerical method
3. Results and discussion
4. Conclusions

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Conclusions

- N-S equations based on homogeneous mixture model could be effective method to analyze cavitation
- The in-house code has been successfully developed to capture cavitation phenomena including full compressible effects, non-condensable gas, buoyancy force, temperature difference effects, and ventilated cavitation.
- Free surface was effectively handled by VOF method
- The 6 DOF motion analysis has been done by using chimera grid.
- Also, the in-house code using full fluid equations, not using homogeneous mixture model, has been also successfully developed.

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Thank you so much!

